<b>M1.</b> (a)	(i)	Appreciates <i>pV</i> should be constant for isothermal change (by working or statement) $W = p\Delta V$ is TO		
		Allow only products seen where are approximately 150 for 1 mark Penalise J as unit here		
			M1	
		Demonstrates $pV$ = constant using 2 points (on the line) set equal to each other or conclusion made or <b>shows</b> that for V doubling that <i>p</i> halves (worth 2 marks)		
		need to see values for p and V		
		Accept statement that products are slightly different so not quite isothermal		
			A1	
		Demonstrates $pV$ = constant using 3 points (on the line) with conclusion		
		Products should equal 150 to 2 sf		
		Accept statement that products are slightly different so not quite isothermal		
			A1	
	(ii)	Adiabatic <u>therefore</u> no heat transfer <b>or</b> Adiabatic <u>therefore</u> Q = 0		
			B1	
		Work is done <u>by</u> gas <u>therefore</u> <i>W</i> is <u>negative</u> <b>or</b> Work is done <u>by</u> gas <u>therefore</u> energy is removed from the system		
			B1	
		$\Delta U$ is negative <u>therefore</u> internal energy of gas decreases <b>or</b> energy is removed from the system <u>therefore</u> internal energy of gas decreases or work done by the gas <u>so</u> internal energy decreases		
		$-\Delta U = -W \text{ or } \Delta U = -W$		
			B1	

Uses pV/T = constant or uses pV=nRT or uses (iii) pV = NkTe.g. makes T subject or substitutes into an equation with  $p_A$  and  $V_A$  or  $p_c$  and  $V_c$  (condone use of n = 1) or  $(pV)_{A}$ their  $(pV)_c$  $V_a$  read off range = 2.5 to 2.6 (× 10<sup>-4</sup>)  $p_A = 600 \times 10^3$  $V_c$  read off range = 8.5 to 8.6 (× 10<sup>-4</sup>)  $p_c = 140 \times 10^3$ C1 Correct substitution of coordinates (inside range) into  $(pV)_A$ (pV)\_ With consistent use of powers of 10  $(pV)_{A}$  range is 150 to 156 and  $(pV)_{C}$  range is 119 to 120.4 C1 1.2(5) Allow range from 1.2 to 1.3 Accept decimal fraction : 1 A1

3

Energy per large square = 10(J) or <u>states</u> that work done is equal to area under curve (between A and B) or energy per small square = 0.4(J)

or square counting seen on correct area

(b)

Must be clear that area represents energy either by subject of formula or use of units on 10 or 0.4

Alternative: W = area of a trapezium(with working) or  $W = P_{mean} \times \Delta V$  or  $W = 450 \times 10^3 \times 2.5 \times 10^{-4}$ or W = area of a rectangle + area of a triangle (with working)

			[14]
		A1	3
	= 0.061 × 10 <sup>-3</sup> or 6.06 × 10 <sup>-5</sup> (m <sup>3</sup> )		
		C1	
	Condone E = 114 (J) <b>or</b> temperature = 291(K)		
	or $\left(\frac{v}{t}\right) = \frac{\Delta \theta}{\rho c \Delta \theta}$ and correct substitution seen		
	or allow V / t = $1.67 \times 10^{-3} \div 1100$		
		C1	
	or their energy $\div$ (c $\Delta T$ ) or their energy $\div$ 68400		
	or number of cycles per s = 40 or (Mass per second =) 114 ÷ 68400 in rearranged form		
(c)	(Total energy removed per s =) 4560 (J)		
		B1	2
	States that actual work done would be lower because of curvature of line		
	Number of small squares = 263 to 287 seen and $(W)$ = number of squares × area of one square (using numbers) Range = 105 to 115 (J)		
	number of squares × area of one square (using numbers) Range = 105 to 115 (J) Or		
	Number of large squares = 10.5 to 11.5 seen and ( $W$ ) =		

**M2.**(a) (i) Clear statement that for isothermal pV =constant or  $p_1V_1 = p_2V_2$  Applies this to any 2 points on the curve AB  $\checkmark$ e.g.  $1.0 \times 10^5 \times 1.2 \times 10^{-3} = 4.8 \times 10^5 \times 0.25 \times 10^{-3} 120 = 120$ Allow pV = c applied to intermediate points **estimated** from graph e.g.  $V = 0.39 \times 10^{-3}$ ,  $p = 3 \times 10^5$ 

2

B1

(ii) 
$$W = p \Delta v$$
  
= 4.8 × 10<sup>5</sup> × (0.39 - 0.25) × 10<sup>-3</sup>  
= 67 J \checkmark

(b)

	Q/J	W/J	∆U/J	
process $A \rightarrow B$	-188	-188	0	1
process $B \rightarrow C$	+235	(+)67	(+)168	~
process $C \rightarrow A$	0	+168	-168	1
whole cycle	+47	+47	0	1

Any horiz line correct up to max 3 Give CE in  $B \rightarrow C$  if ans to ii used for W If no sign take as +ve

 $\eta_{overall} = 47 / 235 = 0.20 \text{ or } 20\%$  🖌 (C)

(d) Isothermal process would require engine to run very slowly / be made of material of high heat conductivity 🖌 Adiabatic process has to occur very rapidly / require perfectly insulating container / has no heat transfer 🖌 Very difficult to meet both requirements in the same device  $\checkmark$ Very difficult to arrange for heating to stop exactly in the right place (C) so that at end of expansion the curve meets the isothermal at A 🖌

Page 5

Do not credit bald statement to effect adiabatic / isothermal process not possible - must give reason Ignore mention of valves opening / closing, rounded corners, friction, induction / exhaust strokes wtte

max 2

max 3

МЗ.

 (i) Indicated work per cylinder = area of loop ✓ [either stated explicitly or shown on the Figure e.g. by shading or ticking squares or subsequent correct working.]

appropriate method for finding area e.g. counting squares  $\checkmark$ correct scaling factor used [to give answer of 470 J ± 50 J]  $\checkmark$ indicated power = 4 × 0.5 × (4100/60) × 470 = 64 kW  $\checkmark$ 

(ii) (Fuel flow rate = 0.376/100 = 0.00376 litre s<sup>-1</sup>)

Input power (= c.v. × fuel flow rate)

= 38.6 × 10° × 0.00376 √

(= 145 kW)

 $\eta_{\text{overall}}$  = brake power/input power  $\checkmark$  seen or implied from correct subsequent working

(b) Power expended in overcoming friction

in (all) the bearings / between piston & cylinder  $\checkmark$ and / or in circulating oil / cooling water  $\checkmark$ and / or driving auxiliaries (e.g. fuel injection pump)  $\checkmark$ 

(c) Represents the induction <u>and</u> exhaust (strokes) (which take place at nearly atmospheric pressure). √

4

3

1

correct scaling factor (1) (to give answer  $\approx$  500 kJ)

- (ii) P (= work done per kg x fuel flow rate)
  = 500 (kJ) × 9.9 (kgs<sup>1</sup>) = 5000kW (1)
  (4950kW)
- (iii) (output power = indicated power friction power)  $P_{out} = 4950 - 430 = 45(20) \, kW$  (1) (use of P = 5000 gives  $P_{out} = 45(70)kW$ ) (allow C.E. for values of P in (ii))

(b) (i) 
$$P_{in}$$
 (= fuel flow rate × calorific value)  
=  $0.30 \times 44 \times 10^{\circ} = 13(.2) \times 10^{\circ}W$  (1)

efficiency = 
$$\frac{4520 \times 10^3}{13.2 \times 10^6} = 34\%$$
  
(allow C.E. for value of  $P_{out}$  in (a) (iii) and  $P_m$  in (b) (i))

j	2	

2

5

[7]

**M5.** (a)  $T_{H} = 273 + 820 = 1093$  (K),  $T_{c} = 273 + 77 = 350$  (K) (1)

efficiency = 
$$\frac{T_H - T_C}{T_H} = \frac{1093 - 350}{1093} = 0.68 \text{ or } 68\%$$
 (1)

(b) rotational speed of output shaft =  $\frac{1800}{2 \times 60}$  = 15 rev s<sup>-1</sup> (1) (work output each cycle = 380 J, 2 rev = 1 cycle in a 4 stroke engine) indicated power = 15 × 190 = 5.7 kW (1)

- (c) power lost (= indicated power –actual power) = 5.7 4.7 = 1.0 kW (1) (allow C.E. for incorrect value from (b))
- (d) energy supplied per sec (= fuel flow rate x calorific value)

$$= \frac{2.1 \times 10^{-2}}{60} \times 45 \times 10^{6} = 16 \ kW \ (15.8 \ kW) \ (1)$$

(e) efficiency =  $\frac{\text{net power output}}{\text{power input}} = \frac{4.7}{16} = 0.29 \text{ or } 29 \%$  $\frac{4.7}{15.8} = 0.30 \text{ or } 30\%$ 

(allow C.E. for value from (d))

[7]

2

1

1